

THE STRUCTURE OF SEPARATED FLOW AND HEAT TRANSFER IN A ROUND TUBE WITH SINGLE DETACHED DIAPHRAGM

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Results of numerical study of the flow structure and turbulent heat transfer in a round tube with one diaphragm of height h and some clearance c between the tube and diaphragm are presented. The clearance between the diaphragm and tube wall was varied within $A = c/h = 0 \div 0.33$ (Fig.1).

Calculations were performed in the framework of the model of incompressible liquid based on a system of stationary Reynolds averaged Navier-Stokes and energy equations (RANS). The main instrument of this study is universal computational complex FLUENT. The problem statement is two-dimensional; the flow is stationary and axisymmetric. The turbulence model $k-\omega$ SST is chosen as the most appropriate for calculation of detached flows. The Reynolds number calculated by the tube diameter and mean-mass velocity was constant $Re = 27500$.

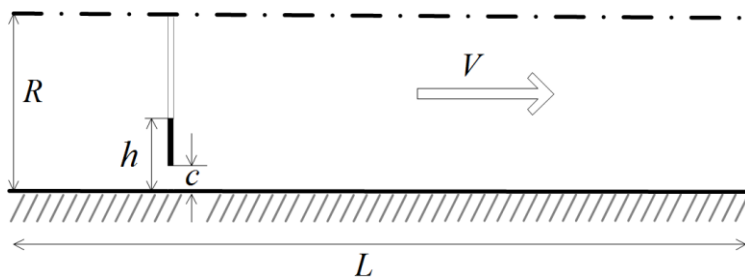


Fig. 1. The scheme of computational domain

It is shown that the near-wall jet formed in a clearance deforms the detached flow behind the diaphragm. With the large scales of the jet ($A > 0.2$) reattachment of the flow does not occur. Due to destruction of the recirculation zone, the level of kinetic energy is decreased and carried away downstream.

It is determined that a rise of a distance between the diaphragm and tube wall reduces maximal heat transfer significantly.

The integral characteristics demonstrate that for clearance $c/h = 0.033$ and 0.067 , heat transfer is enhanced due to an increased heat removal on the tube wall under the diaphragm.

The pressure losses are reduced by the factor of 3 at a change in the size of near-wall jet from $c/h = 0$ to 0.33 . At that, thermal enhancement factor increases by 30% (Fig.2).

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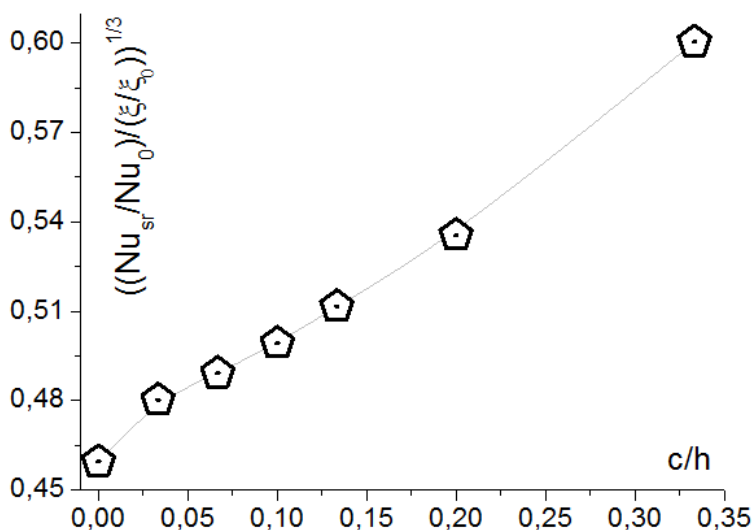


Fig. 2. Thermal enhancement factor

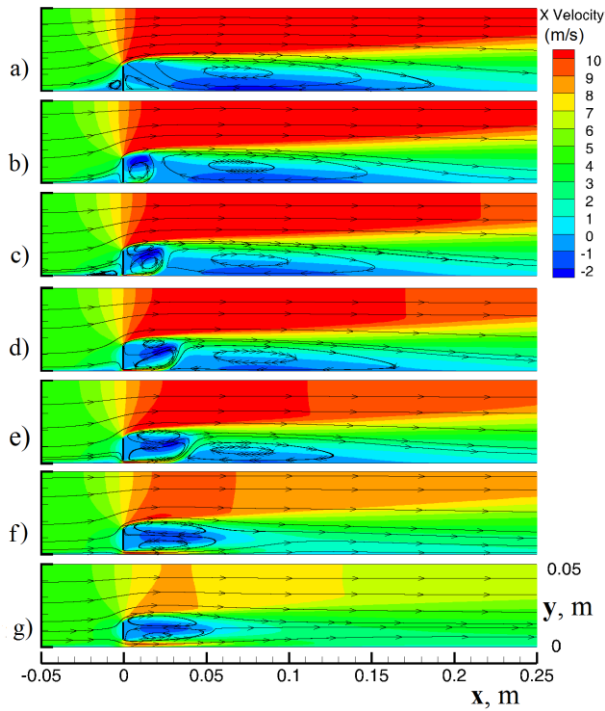


Fig. 3. The stream lines and the longitudinal component of the velocity field in the area of the diaphragm. a) $c/h = 0$, attached diaphragm; b) 0.03; c) 0.07; d) 0.1; e) 0.13; f) 0.2; g) 0.33.

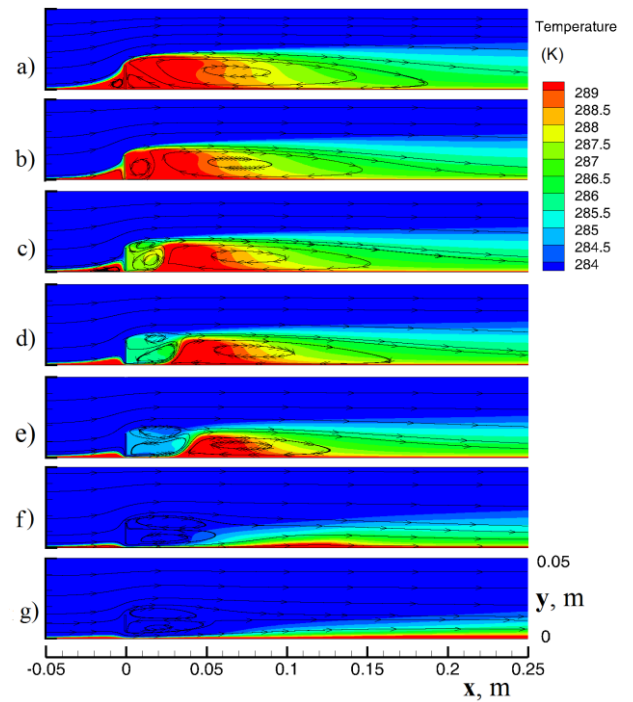


Fig. 4. The stream lines and temperature field around the diaphragm. a) $c/h = 0$, attached diaphragm; b) 0.03; c) 0.07; d) 0.1; e) 0.13; f) 0.2; g) 0.33.

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REFERENCES

1. Liou T.M., Chen S.H. Turbulent heat and fluid flow in a passage disturbed by detached perforated ribs of different heights. *Int. J. Heat and mass Transfer*, vol. 41, pp. 1795–1806, 1998.
2. Tsia J.P., Hwang J.J. Measurements of heat transfer and fluid flow in a rectangular duct with alternate attached detached rib-arrays. *Int J Heat Mass Transf.* – 1999. V. 42, P. 2071–2083.
3. Ahn J, Lee J.S. Large eddy simulation of flow and heat transfer in a channel with a detached rib array. *Int. J. Heat and mass Transfer*, vol. 53, pp. 445–452, 2010.
4. Eiamsa-ard S., Changcharoen W. Numerical Investigation of Turbulent Heat Transfer in Channels with Detached Rib-Arrays// *Heat Transf - Asian Research* 40 (5), 2011.